Seminar of I. Vekua Institute of Applied Mathematics REPORTS, Vol. 32-33, 2006-2007

A CUSPED KIRCHHOFF-LOVE PLATE UNDER THE ACTION OF CONCENTRATED LOADINGS

Jaiani G.

I. Vekua Institute of Applied Mathematics of Iv. Javakhishvili Tbilisi State University

Abstract. The paper deals with a cusped Kirchhoff-Love plate under the action of concentrated bending moment M and concentrated generalized shearing force Q. In the case when the projection of the plate is a half-plane the problem is solved in the explicit form.

Keywords and phrases: Cusped plate, Kirchhoff-Love bending model, concentrated bending moment, concentrated shearing force.

AMS subject classification (2000): 74K20; 35J70

The paper deals with a cusped Kirchhoff-Love plate [1-3] with the flexural rigidity

$$D = D_0 x_2^{\kappa} = D_0 r^{\kappa} \sin^{\kappa} \psi, \quad x_2 \ge 0, \quad D_0, \ \kappa = \text{const}, \quad D_0 > 0, \ \kappa \ge 0.$$

Let at the edge of the plate a concentrated bending moment M and concentrated generalized shearing force Q are applied:

$$M_2(x_1, 0) = -f_0(x_1) = -M\delta(x_1), \qquad (1)$$

$$Q_2^*(x_1,0) = Q_2(x_1,0) - \frac{\partial M_{12}(x_1,0)}{\partial x_1} = -f_1(x_1) = -Q\delta(x_1), \qquad (2)$$

where M_2 and M_{12} denote bending and twisting moments, respectively, Q_2^* and Q_2 denote shearing and generalized shearing forces, correspondingly, $\delta_{x_1}(\varphi)$, $\delta := \delta_0$ is the Dirac Function, i. e.,

$$\delta_{x_1}(\varphi) = \varphi(x_1) = \int_{-\infty}^{+\infty} \varphi(\xi) \delta(\xi - x_1) d\xi.$$

As it is well-known generalized derivative of the Heaviside function

$$Y(\xi) = \begin{cases} +1, \ \xi > 0. \\ 0, \ \xi \le 0, \end{cases}$$

is the Dirac function. Consequently, for the antiderivative $(Q\delta(x_1))^{(-1)}$ we have

$$(Q\delta(x_1))^{(-1)} = QY(\xi) = \begin{cases} Q, \ \xi > 0.\\ 0, \ \xi \le 0. \end{cases}$$

Let

$$r^2 := x_1^2 + x_2^2, \ \psi := \operatorname{arcctg} \frac{x_1}{x_2};$$

 $\rho^2 := (x_1 - \xi)^2 + x_2^2, \quad \theta := \operatorname{arcctg} \frac{x_1 - \xi}{x_2},$

i.e.,

$$\begin{aligned} \xi &= x_1 - x_2 \operatorname{ctg}\theta, \ d\xi = x_2 \frac{d\theta}{\sin^2 \theta}; \\ A(c,b) &:= \int_0^{\pi} \cos(c\theta) \sin^{-b} \theta d\theta, \ b < 1; \ B(c,b) &:= \int_0^{\pi} \sin(c\theta) \sin^{-b} \theta d\theta, \ b < 1; \\ K &:= c D_0 \left(b^2 - c^2 \right) \left[A^2(c, -b) + B^2(c, -b) \right]; \\ \Lambda(c,b) &:= \int_0^{\pi} e^{a\theta} \sin^{-b} \theta d\theta, \ b < 1; \\ a &:= \sqrt{(1 - \kappa \sigma)(\kappa - 1)}, \ c &:= \sqrt{(\kappa \sigma - 1)(\kappa - 1)}, \ b &:= \kappa - 1. \end{aligned}$$

Applying formally well known (see [1] and [2], p. 112) representation of the solution of the bending of the cusped plate under the action of a bending moment and generalized shearing force, when the projection of the plate is a half-plane, for the deflection w we get

$$w(x_{1},x_{2}) = \frac{1}{K} \int_{-\infty}^{+\infty} \left\{ \left[cA(c,-b)M\delta(\xi) - bB(c,-b)QY(\xi) \right] \cos\left(c \operatorname{arcctg} \frac{x_{1}-\xi}{x_{2}}\right) \right\} \\ + \left[cB(c,-b)M\delta(\xi) - bA(c,-b)QY(\xi) \right] \sin\left(c \operatorname{arcctg} \frac{x_{1}-\xi}{x_{2}}\right) \right\} \\ \times \left[(x_{1}-\xi)^{2} + x_{2} \right]^{-\frac{b}{2}} d\xi = K^{-1} \left\{ Mc \left[A(c,-b)\cos\left(c \operatorname{arcctg} \frac{x_{1}}{x_{2}}\right) \right] \\ + B(c,-b)\sin\left(c \operatorname{arcctg} \frac{x_{1}}{x_{2}}\right) \right] (x_{1}+x_{2})^{-\frac{b}{2}} \\ - Qb \int_{0}^{+\infty} \left[B(c,-b)\cos\left(c \operatorname{arcctg} \frac{x_{1}-\xi}{x_{2}}\right) + A(c,-b)\sin\left(c \operatorname{arcctg} \frac{x_{1}-\xi}{x_{2}}\right) \right] \\ \times \left[(x_{1}-\xi) + x_{2}^{2} \right]^{-\frac{b}{2}} d\xi \right\} \\ = K^{-1} \left\{ Mc \left[A(c,-b)\cos(c\psi) + B(c,-b)\cos(c\psi) \right] r^{-b} \\ - Qb \int_{\psi}^{\pi} \left[B(c,-b)\cos(c\psi) + A(c,-b)\sin(c\theta) \right] \frac{x_{2}^{1-b}\sin^{b}\theta}{\sin^{2}\theta} d\theta \right\} \\ = K^{-1} \left\{ Mc \left[A(c,-b)\cos(c\psi) + B(c,-b)\cos(c\psi) \right] r^{-b} \\ - Qbr^{1-b}\sin^{1-b}\psi \int_{\psi}^{\pi} \left[B(c,-b)\cos(c\theta) + A(c,-b)\sin(c\theta) \right] \sin^{b-2}\theta d\theta \right\}$$
(3)

(since the Poisson's ratio $\sigma < \frac{1}{2}$ in the case under consideration we have $\kappa > 2$ and all the integrals in the previous expressions are convergent);

$$w(x_{1}, x_{2}) = \frac{1}{D_{0}b\Lambda(0, -b)} \int_{-\infty}^{+\infty} \left[b^{-1}M\delta(\xi) - \frac{\pi}{2}QY(\xi) + QY(\xi)\theta \right] \rho^{-b}d\xi$$
$$= \frac{M}{D_{0}b^{2}\Lambda(0, -b)} r^{-b} - \frac{Q}{D_{0}b\Lambda(0, -b)} \int_{0}^{+\infty} \left(\frac{\pi}{2} - \theta\right) \rho^{-b}d\xi = \frac{M}{D_{0}b^{2}\Lambda(0, -b)} r^{-b}$$
$$- \frac{Q}{D_{0}b\Lambda(0, -b)} r^{1-b} \sin^{1-b}\psi \int_{\psi}^{\pi} \left(\frac{\pi}{2} - \theta\right) \sin^{b-2}\theta d\theta \text{ for } \kappa = \frac{1}{\sigma}$$
(4)

(in the case under consideration we have $\kappa > 2$ and all the integrals in the previous expressions are convergent);

$$w(x_{1}, x_{2}) = \frac{1}{2a(a^{2} + b^{2}) D_{0}\Lambda(a, -b)}$$

$$\times \int_{-\infty}^{+\infty} \{ [aM\delta(\xi) - bQY(\xi)] e^{a(\pi-\theta)} + [aM\delta(\xi) + bQY(\xi)] e^{a\theta} \} \rho^{-b} d\xi$$

$$= \frac{M}{2a(a^{2} + b^{2}) D_{0}\Lambda(a, -b)} \left[e^{a(\pi-\psi)} + e^{a\psi} \right] r^{-b}$$

$$- \frac{Qb}{2a(a^{2} + b^{2}) D_{0}\Lambda(a, -b)} \int_{0}^{+\infty} \left[e^{a(\pi-\theta)} - e^{a\theta} \right] \rho^{-b} d\xi$$

$$= \frac{M}{2a(a^{2} + b^{2}) D_{0}\Lambda(a, -b)} \left[e^{a(\pi-\psi)} + e^{a\psi} \right] r^{-b}$$

$$- \frac{Qb}{2a(a^{2} + b^{2}) D_{0}\Lambda(a, -b)} \sin^{1-b}\psi \cdot r^{1-b} \int_{\psi}^{\pi} \left[e^{a(\pi-\theta)} - e^{a\theta} \right] \sin^{b-2}\theta d\theta$$
for $1 < \kappa < \frac{1}{\sigma}$
(5)

(in the case under consideration the integral exists only if b > 1, i.e., $\kappa > 2$, while by Q = 0 there remains only the first summand which has a physical sense for $\kappa > 1$).

Using (3)-(5) it is not difficult to get expressions for bending moments and shearing forces and directly to verify that they with (3)-(5) represent the explicit solution of the problem under consideration.

$\mathbf{R} \to \mathbf{F} \to \mathbf{R} \to \mathbf{N} \to \mathbf{C} \to \mathbf{S}$

1. Jaiani G. Bending of a plate with stiffness that varies according to the power law, *Godishnik Vissh. Uchebn. Zaved. Tekhn. Mekh.* **12**, 2 (1977), 15-19, Mechanics (Third Congress, Varna, 1977) (in Russian. English, Bulgarian summary)

2. Jaiani G. Solution of Some Problems for a Degenerate Elliptic Equation of Higher Order and their Applications to Prismatic Shells, *Tbilisi University Press*, 1982 (in Russian, Georgian and English summaries)

3. Timoshenko S., Woinowsky-Krieger S. Theory of Plates and Shells, *Mcgraw-Hill Book Company*, *New York-Toronto-London*, 1959.

Received: 17.11.2006; revised: 20.12.2006; accepted: 27.12.2006.