

ON DISCLOSING THE LOCATION OF ACCIDENTAL GAS ESCAPE
FROM THE COMPLICATED MAIN PIPE-LINE WITH
NON-STATIONARY FLOW AND DETERMINING THE INTENSITY
OF THE ESCAPE

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Abstract

Linear mathematical model of gas flow in the main pipe-line is considered. The method of disclosing the location of accidental gas escape from the complicated main with non-stationary gas flow and determining the intensity of the escape is described and appropriate formulas are received

The efficiency of the presented method is illustrated on a test example .

Key words and phrases: boundary condition, pipe-line, gas flow, accidental escape .

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The solution of the problem of disclosing the location of an accidental gas escape from the main pipe-line is known not only for simple pipe-line [1] but also for the complicated one [2] with stationary flow. The problem (with determining the intensity of the escape) is solved also with non-stationary flow for simple pipe-line [3]. In the presented work off-shoots from the main pipe-line are taken into account.

Distribution of the gas pressure along the main $u = u(x, t)$ is described by the following equation

$$\frac{\partial u}{\partial t} = a^2 \frac{\partial^2 u}{\partial x^2} + \sum_{k=1}^m M_k \delta(x - x_k) + q \delta(x - x^*) \sigma(t - t_0), \quad 0 < x < L \quad t > 0 \quad (1)$$

where $a^2 = const > 0$, m is number of off-shoots, $M_k (k = \overline{1, m})$ - expenses of gas in the off-shoots, $x_k (k = \overline{1, m})$ -coordinates of points of off-shoots, q -coordinate of the point of escape, t_0 - the moment of the beginning of the escape, $\delta(\cdot)$ -Dirac function, $\sigma(\cdot)$ -Heaviside function [1], L - the length of the main x^* is considered the point of off-shoot.

With equation (1) let us consider the following initial and boundary conditions:

$$u(x, 0) = Q(x), \quad 0 \leq x \leq L, \quad (2)$$

$$\frac{\partial u}{\partial x} = g_1 \quad x = 0, \quad t > 0, \quad (3)$$

$$\frac{\partial u}{\partial x} = g_2 \quad x = L, \quad t > 0, \quad (4)$$

where $Q(x)$ is the initial distribution of pressure, which can be represented in the following form:

$$Q(x) = \frac{S - R}{L}x + R.$$

Here R and S ($R \geq S$) are gas pressures at the beginning and ending of the main, respectively, at the initial moment.

Sought for values are: coordinate of the escape location x^* and intensity of the escape q . Therefore two additional conditions are needed. For this purpose we will use the values \bar{R} and \bar{S} received by measuring pressure at the beginning and ending of the main at moment of time T :

$$u(0, T) = \bar{R}, \quad (5)$$

$$u(L, T) = \bar{S}. \quad (6)$$

It is known [4] that the solution of the problem (1)-(4) can be represented by the following formula:

$$u(x, t) = \int_0^t \int_0^L G(x, \xi, t - \tau) \omega(\xi, \tau) d\xi d\tau, \quad (7)$$

where G is Green function

$$G(x, \xi, t - \tau) = \frac{1}{L} \left[1 + 2 \sum_{n=1}^{\infty} \cos \frac{n\pi x}{L} \cos \frac{n\pi \xi}{L} e^{-\frac{n^2 a^2 \pi^2}{L^2} (t - \tau)} \right].$$

Here

$$\omega(x, t) = Q(x)\delta(t) - a^2\delta(x)g_1 + a^2\delta(L-x)g_2 + \sum_{k=1}^m M_k\delta(x-x_k) + q\delta(x-x^*)\sigma(t-t_0).$$

Taking into account the last equality, formula (7) will have the following form:

$$\begin{aligned} u = & \int_0^L G(x, \xi, t) Q(\xi) d\xi - a^2 g_1 \int_0^t G(x, 0, t - \tau) d\tau + a^2 g_2 \int_0^t G(x, L, t - \tau) d\tau \\ & + \sum_{k=1}^{\infty} M_k \int_0^t G(x, x_k, t - \tau) d\tau + q \int_{t_0}^t G(x, x^*, t - \tau) d\tau \end{aligned} \quad (8)$$

If we retain only the first item in the expression of Green function, taking into account formula (8), equalities (5),(6) will be rewritten as follows:

$$\begin{aligned} \bar{R} = & \frac{2q}{LP} (1 - e^{-P(T-t_0)}) \cos \frac{\pi x^*}{L} + \frac{T-t_0}{L} q - \frac{a^2 g_1}{L} \left[T + \frac{2}{P} (1 - e^{-PT}) \right] \\ & + \frac{a^2 g_2}{L} \left[T - \frac{2}{P} (1 - e^{-PT}) \right] + \frac{R+S}{2} + \frac{4(R-S)}{\pi^2} e^{-PT} \\ & + \frac{2}{LP} (1 - e^{-PT}) \sum_{k=1}^m M_k \cos \frac{\pi x_k}{L} + \frac{T}{L} \sum_{k=1}^m M_k, \end{aligned} \quad (9)$$

$$\begin{aligned} \bar{S} = & -\frac{2q}{LP} (1 - e^{-P(T-t_0)}) \cos \frac{\pi x^*}{L} + \frac{T-t_0}{L} q - \frac{a^2 g_1}{L} \left[T + \frac{2}{P} (e^{-PT} - 1) \right] \\ & + \frac{a^2 g_2}{L} \left[T + \frac{2}{P} (1 - e^{-PT}) \right] + \frac{R+S}{2} - \frac{4(R-S)}{\pi^2} e^{-PT} \\ & - \frac{2}{LP} (1 - e^{-PT}) \sum_{k=1}^m M_k \cos \frac{\pi x_k}{L} + \frac{T}{L} \sum_{k=1}^m M_k, \end{aligned} \quad (10)$$

where $P = a^2 \pi^2 / L^2$.

Equalities (9) and (10) form the system of equations with two unknown values: x^* and q . Summing up these equations item by item we will have:

$$\bar{R} + \bar{S} = \frac{2(T-t_0)}{L} q - \frac{2a^2 T}{L} (g_1 - g_2) + R + S + \frac{2T}{L} \sum_{k=1}^m M_k.$$

From this

$$q = \frac{L(\bar{R} + \bar{S} - R - S) + 2a^2 T(g_1 - g_2) - 2T \sum_{k=1}^m M_k}{2(T-t_0)}. \quad (11)$$

In order to calculate x^* , equations (9) or (10) might be used. In particular, from equation (10) we will have:

$$\begin{aligned} x^* = & \frac{L}{\pi} \arccos \left\{ \frac{1}{2q(1 - e^{-P(T-t_0)})} \left[P(T-t_0)q + \frac{LP}{2}(R+S) \right. \right. \\ & - LP\bar{S} + \frac{4LP(R-S)}{\pi^2} e^{-PT} + PT \sum_{k=1}^m M_k + Pa^2 T(g_2 - g_1) \\ & \left. \left. + 2 \left(a^2 g_1 + a^2 g_2 - \sum_{k=1}^m M_k \cos \frac{\pi x_k}{L} \right) (1 - e^{-PT}) \right] \right\}, \end{aligned} \quad (12)$$

where q is determined by formula (11).

To illustrate the presented method, let us consider a problem, the solution of which is known beforehand. Suppose, the endings of the pipe-line are closed.

$$g_1 = g_1 = 0 \quad (13)$$

and initial pressure is the same along the pipe-line:

$$S = R. \quad (14)$$

Also, let us assume, that there are two off-shoots at points located at the equal distance from the beginning and ending of the main:

$$x_1 = \frac{1}{4}L, \quad x_2 = \frac{3}{4}L \quad (15)$$

and with equal intensity:

$$M_1 = M_2 = M. \quad (16)$$

Apart from the above said, suppose at a T moment if time, after measuring pressure, it was found that pressures at the beginning and ending of the main are the same:

$$\bar{S} = \bar{R}. \quad (17)$$

It is obvious that under such conditions the place of gas escape will be the middle point of the main. Indeed, taking into account (13)-(17) in formulas (11),(12) we will have:

$$q = \frac{2L(\bar{R} - R) - 4TM}{2(T - t_0)}, \quad (18)$$

$$\begin{aligned} x^* &= \frac{L}{\pi} \arccos \left\{ \frac{T - t_0}{[2L(\bar{R} - R) - 4TM] (1 - e^{-P(T-t_0)})} \left[PL(\bar{R} - R) \right. \right. \\ &\quad \left. \left. - 2PTM + LPR - LP\bar{R} + 2PTM - 2M \left(\cos \frac{\pi}{4} + \cos \frac{3\pi}{4} \right) (1 - e^{-PT}) \right] \right\} \\ &= \frac{L}{\pi} \arccos \left\{ \frac{T - t_0}{[2L(\bar{R} - R) - 4TM] (1 - e^{-P(T-t_0)})} \cdot 0 \right\} \\ &= \frac{L}{\pi} \arccos 0 = \frac{L}{\pi} \cdot \frac{\pi}{2} = \frac{L}{2}. \end{aligned}$$

R E F E R E N C E S

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